

11th HSTAM International Congress On Mechanics
May 27-30 , Athens, Greece

***Study of the Static Response of a Coated
Microbubble Under the AFM:
Numerical & Asymptotic Analysis***

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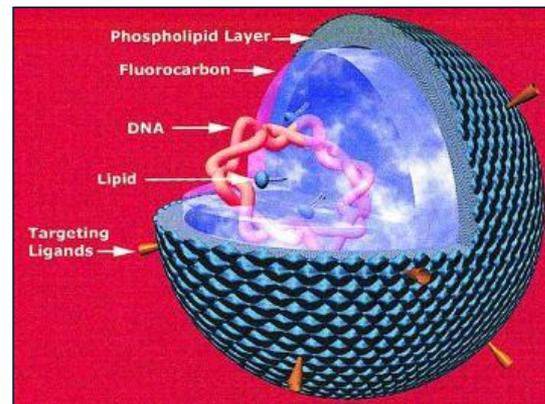
University of Thessaly, Department of Mechanical Engineering,
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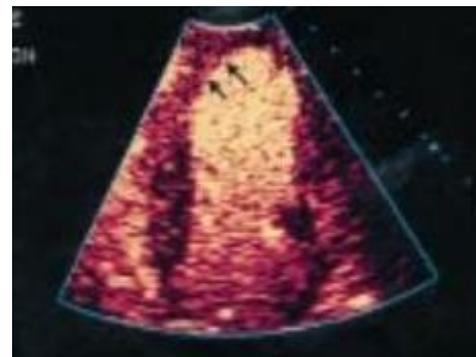
Outline

- Introduction
- Mathematical Modeling
- Numerical Analysis
- Results
- Conclusions

- Elastic coating (Polymer or Lipid)
- Initial Diameter $\sim 2\text{-}5\ \mu\text{m}$
- Initial Thickness $\sim 1\text{-}30\ \text{nm}$
- Strong Backscattered Acoustic Signal
- Biocompatibility

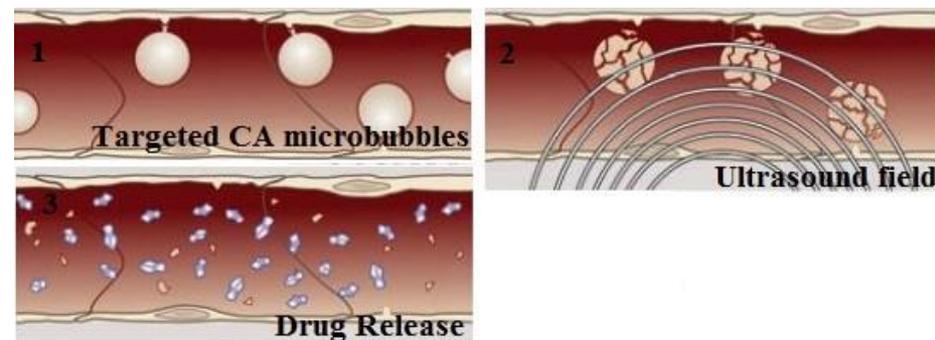


- Medical Imaging-Diagnostic Applications (Heart, Kidney, Liver)



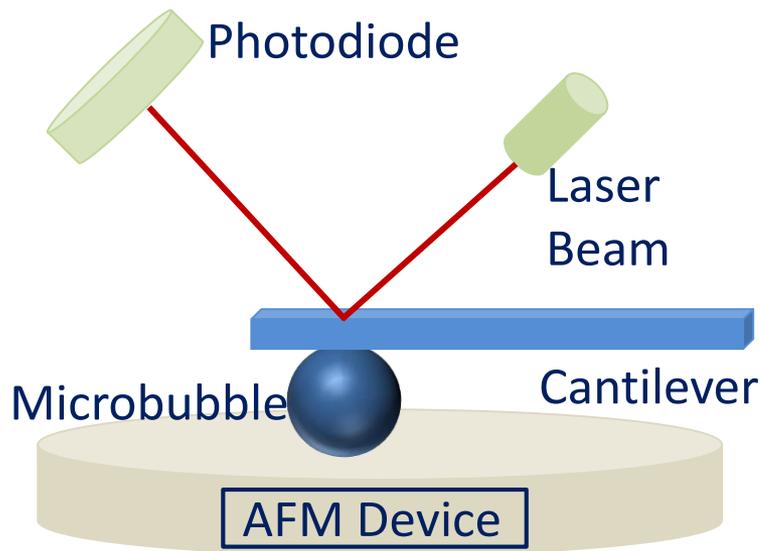
[Blomley et al., *Br. Med. J.*, 2001]

- Treatment with microbubbles- Drug or gene delivery



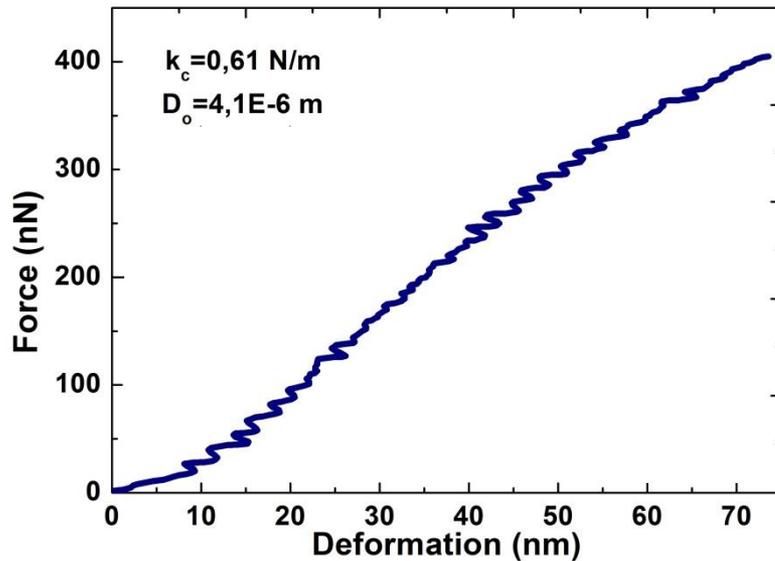
[Ferrara et al., *Annu. Rev. Biomed. Eng.*, 2007]





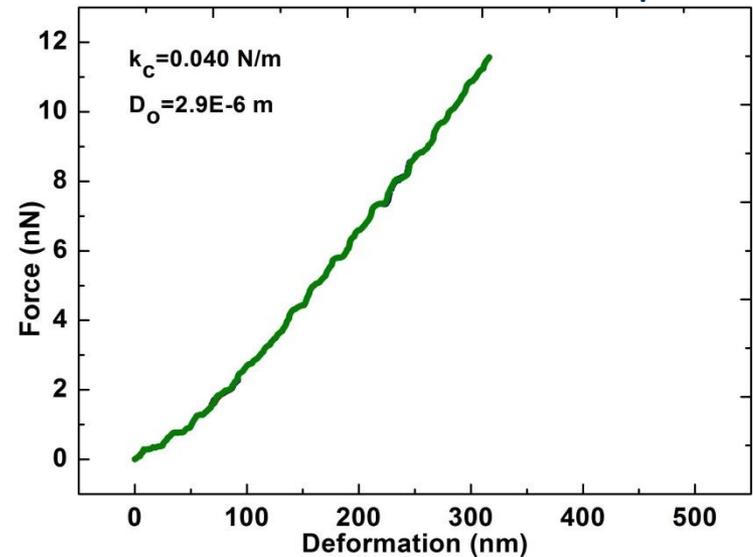
Experimental Investigation of the Mechanical Properties with the Atomic Force Microscope (AFM).

Microbubble covered with polymer



[Glynos et al., *Langmuir*, 2009]

Microbubble covered with lipid



[Buchner Santos et al., *Langmuir*, 2012]

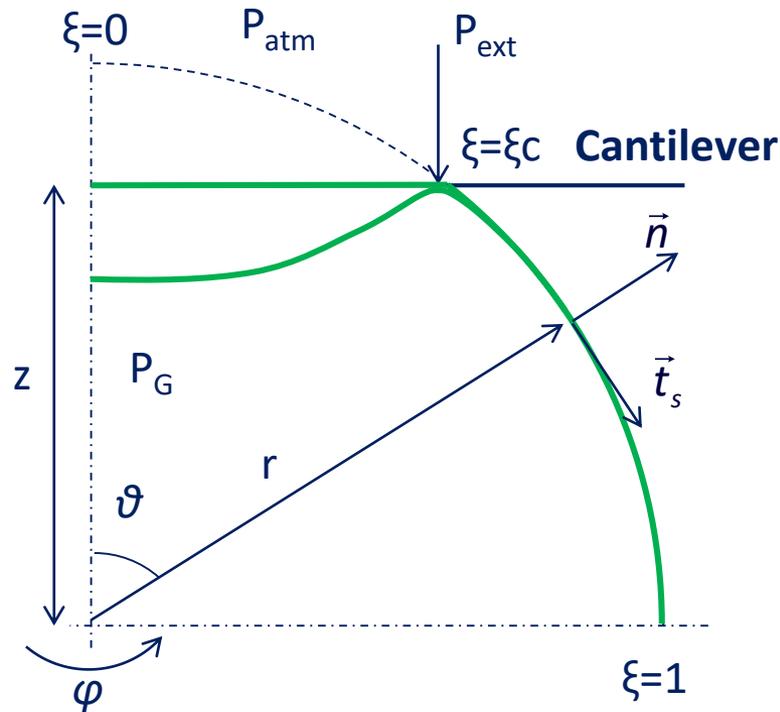


Scope of the Present Work

- Simulation of the static response of CA microbubbles.
- Comparison with AFM experimental measurements.
- Estimation of the elastic properties (Young's & bending modulus).
- Optimization of the design of microbubbles



Classic Contact Problem (Polymeric Bubbles)



Deformation= $z(\xi_c)$

Comparison against experiments

Normal and Tangential Force Balance

$$\Delta \vec{F} = \Delta \vec{P} \Rightarrow \begin{cases} \Delta \vec{F}_N = (P_G - P_{atm}) \underline{l} \cdot \vec{n} \\ \Delta \vec{F}_s = \vec{0} \end{cases}$$

$$\Delta \vec{F} = -\vec{\nabla}_s \cdot (\tau_{ss} \vec{t}_s \vec{t}_s + \tau_{\varphi\varphi} \vec{t}_\varphi \vec{t}_\varphi + q \vec{t}_s \vec{n}) + \sigma \vec{\nabla}_s \cdot \vec{n}$$

$$\vec{q} = \vec{\nabla}_s \cdot \underline{m} \cdot (\underline{l} - \vec{n} \vec{n}) \quad ; \quad q^+(\xi_c) + q^-(\xi_c) = P_{ext}$$

Kinematic Condition

$$z_\xi = 0 \quad \text{at} \quad \xi = \xi_c$$

Isothermal Gas Compression

$$P_{G0} V_0^\gamma = P_G V^\gamma$$

Constitutive Equation: Hook's Law

$$\tau = G_s \frac{1+\nu}{1-\nu} (\lambda^2 - 1)$$

$$m_i = k_b (K_i + \nu K_j) / \lambda_j, \quad K_i \equiv \lambda_i k_i - k_i^R$$

Effect of pre-stress

$$\lambda_i = \frac{dS}{dS^{SF}}, \quad \text{where} \quad \lambda_i^0 = \frac{dS^0}{dS^{SF}} \neq 1$$

Boundary Conditions

$$r_\xi = \vartheta_{\xi\xi} = 0 \quad \text{at} \quad \xi = 0 \text{ and } 1$$

[Pozrikidis, *J. Fluid Mech.*, 2001]

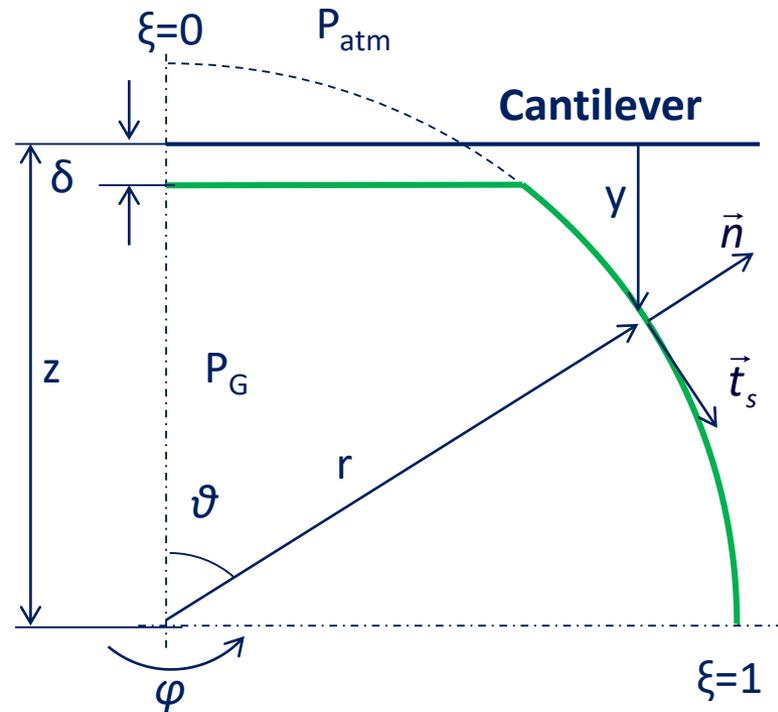
[Tsigliferis & Pelekasis, *Phys. Fluids*, 2011]

[Barthès-Biesel et al., *J. Fluid Mech.*, 2002]

[Updike & Kallins, *Trans. ASME*, 1970, 1972, 1972]



Adhesive Potential-Bubble Covered With Elastic Shell



Deformation = $z(F=0) - z$



Comparison against experiments

Normal and Tangential Force Balance

$$\Delta \vec{F} = \Delta \vec{P} \Rightarrow \Delta \vec{F}_N = (P_G - P_{atm}) \underline{\underline{I}} \cdot \vec{n} \quad \& \quad \Delta \vec{F}_s = \vec{0}$$

$$\Delta \vec{F} = -\vec{\nabla}_s \cdot (\tau_{ss} \vec{t}_s \vec{t}_s + \tau_{\varphi\varphi} \vec{t}_\varphi \vec{t}_\varphi + q \vec{t}_s \vec{n}) + \vec{\nabla} \cdot [(W + \sigma) \vec{n} \vec{n}]$$

$$\vec{q} = \vec{\nabla}_s \cdot \underline{\underline{m}} \cdot (\underline{\underline{I}} - \vec{n} \vec{n})$$

$$W = W(y) = W_0 \left[\left(\frac{\delta_A}{y} \right)^4 - 2 \left(\frac{\delta_A}{y} \right)^2 \right]$$

Isothermal Gas Compression

$$P_{G0} V_0^y = P_G V^y$$

Constitutive Law: Mooney-Rivlin

$$\tau_{ij} = \frac{G_s}{3\lambda_i \lambda_j} \left[\lambda_i^2 - \frac{1}{\lambda_i^2 \lambda_j^2} \right] \left[1 + b(\lambda_j^2 - 1) \right]$$

$$m_i = k_b (K_i + \nu K_j) / \lambda_j, \quad K_i \equiv \lambda_i k_i - k_i^R$$

Effect of pre-stress

$$\lambda_i = \frac{dS}{dS^{SF}}, \quad \text{where } \lambda_i^0 = \frac{dS^0}{dS^{SF}} \neq 1$$

Boundary Conditions

$$r_\xi = \vartheta_{\xi\xi} = 0 \quad \text{at } \xi = 0 \text{ and } 1$$



Adhesive Potential-Free Bubble

Normal Force Balance

$$P_G - P_{atm} = 2H(\sigma - W) + \frac{\partial W}{\partial n}$$

$$W = W(y) = W_o \left[\left(\frac{\delta_A}{y} \right)^4 - 2 \left(\frac{\delta_A}{y} \right)^2 \right]$$

Arc-Length

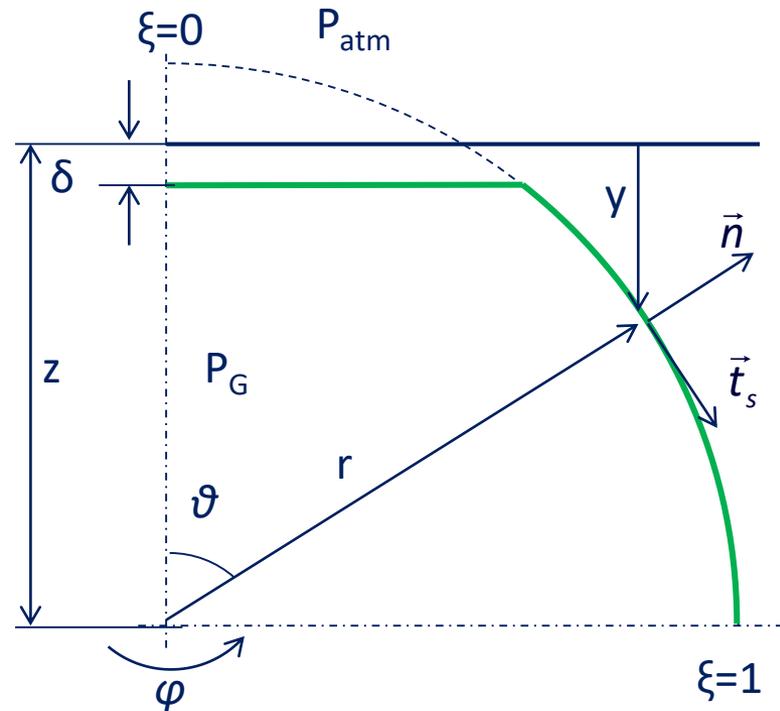
$$ds^2 = (rd\vartheta)^2 + dr^2$$

Isothermal Gas Compression

$$P_{G0} V_0^\gamma = P_G V^\gamma$$

Boundary Conditions

$$r_\xi = \vartheta_{\xi\xi} = 0 \text{ at } \xi = 0 \text{ and } 1$$



Deformation = $z(F=0) - z$



Comparison against experiments

[Chamakovs, Kavousakis & Papathanasiou, *Soft Matter*, 2013]



Finite Elements

Weak Form-Classic contact problem

$$R_1 = \int_0^1 \left[(k_s \tau_{ss} + k_\varphi \tau_{\varphi\varphi} + \sigma 2H + P(\xi) + P_{atm} - P_G) B_i r \sin \vartheta s_\xi - \frac{\sigma (B_{i,\xi\xi} s_\xi - B_{i,\xi} s_{\xi\xi})}{s_\xi^2} - \frac{m_\varphi B_{i,\xi} \sigma_\xi}{s_\xi} \right] d\xi + M_{BT}$$

$$R_2 = \int_0^1 \left[\tau_{ss} B_{i,\xi} \sigma + B_i \sigma_\xi \tau_{\varphi\varphi} + \sigma m_s (k_{s,\xi} B_i + k_s B_{i,\xi}) + k_s m_\varphi B_i \sigma_\xi \right] d\xi + N_{BT}$$

$$R_3 = P_G V^V - P_{atm} V_0^V$$

where

$$\xi = 0: (\vartheta = 0) \quad M_{BT} = 0 \text{ and } N_{BT} = 0$$

$$\xi = 1: \left(\vartheta = \frac{\pi}{2} \right) \quad M_{BT} = \frac{\sigma m_s B_{i,\xi}}{s_\xi} - \sigma q B_i \text{ and}$$

$$N_{BT} = -[k_s m_s + \tau_{ss}] \sigma B_i$$

$$s_\xi = \frac{ds}{d\xi} = \sqrt{r^2 \vartheta_\xi^2 + r_\xi^2} \quad ; \quad \sigma = r \sin \vartheta \quad ; \quad \sigma_\xi = \frac{d\sigma}{d\xi} = r_\xi \sin \vartheta + r \vartheta_\xi \cos \vartheta$$



Finite Elements

Weak Form-Adhesive Potential with Elasticity

$$R_1 = \int_0^1 \left[\left(k_s \tau_{ss} + k_\varphi \tau_{\varphi\varphi} + (\sigma - W) 2H + P_{atm} - P_G \right) B_i r \sin \vartheta s_\xi - \frac{\sigma (B_{i,\xi\xi} s_\xi - B_{i,\xi} s_{\xi\xi})}{s_\xi^2} - \frac{m_\varphi B_{i,\xi} \sigma_\xi}{s_\xi} + \frac{\partial W}{\partial n} \right] d\xi + M_{BT}$$

$$R_2 = \int_0^1 \left[\tau_{ss} B_{i,\xi} \sigma + B_i \sigma_\xi \tau_{\varphi\varphi} + \sigma m_s (k_{s,\xi} B_i + k_s B_{i,\xi}) + k_s m_\varphi B_i \sigma_\xi + \frac{\partial W}{\partial s} \right] d\xi + N_{BT}$$

$$R_3 = P_G V^y - P_{atm} V_0^y$$

Weak Form-Adhesive Potential for a Free Bubble

$$R_1 = \int_0^1 \left[2H(\sigma - W) + P_{atm} - P_G + \frac{\partial W}{\partial n} \right] B_i r \sin \vartheta s_\xi d\xi$$

$$R_2 = \int_0^1 \left[S_{\max} - (r^2 \vartheta_\xi^2 + r_\xi^2)^{0.5} \right] B_i r \sin \vartheta s_\xi d\xi$$

$$R_3 = P_G V^y - P_{atm} V_0^y$$



Finite Elements

- Method of Weighted Residuals:

$$\underline{\underline{J}} \cdot \underline{\underline{c}} = \underline{\underline{R}}$$

$\underline{\underline{J}}$: Jacobian Matrix
 $\underline{\underline{c}}$: Unknown coefficients
 $\underline{\underline{R}}$: Residuals

- Basis Functions:
B-Cubic Splines

$$B_i(t_j) = \begin{cases} 4, & j=1 \\ 1, & j=i \pm 1 \\ 0, & j=i \pm 2 \end{cases}$$

- Calculation of Integrals:

4 Points Gauss Quadrature

- Solution of Linear System:

Newton-Raphson Iterations
Simple or Arc-Length Continuation

- Validation:

Calculation of the critical buckling load and comparison with previous analytical or experimental results.

[Pelekasis, Manolis, Tsamopoulos, *Phys. Fluids*, 1990]

[Tsiglkifis & Pelekasis, *Phys. Fluids*, 2011]

[Lytra & Pelekasis, *Fluid Dyn. Res.*, 2014]



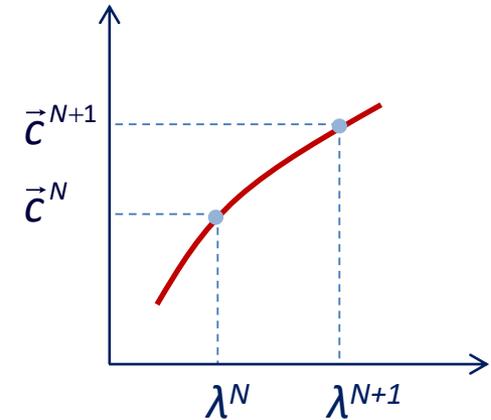
Continuation of the solution

Simple Continuation:

$$\lambda^{N+1} = \lambda^N + \Delta\lambda$$

Initial guess for the

new solution (N+1): $\vec{c}^{N+1} = \vec{c}^N$



Arc-Length Continuation:

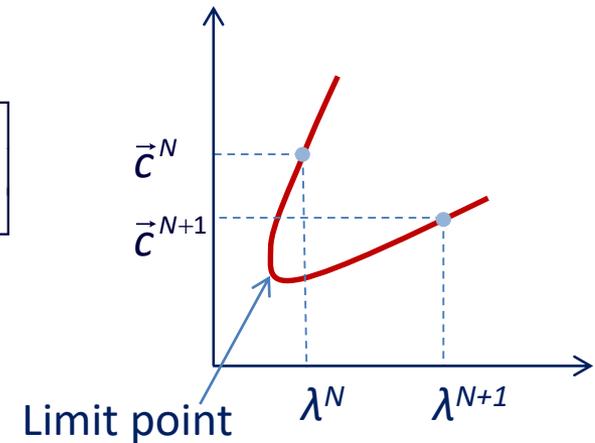
$$\begin{bmatrix} [J] & \vec{R}_\lambda \\ \vec{N}_c & N_\lambda \end{bmatrix} \begin{bmatrix} \vec{c} \\ \lambda \end{bmatrix} = \begin{bmatrix} \vec{R} \\ N \end{bmatrix} \quad \text{and} \quad \begin{bmatrix} [J] & \vec{R}_\lambda \\ \vec{N}_c & N_\lambda \end{bmatrix} \begin{bmatrix} \partial\vec{c}/\partial s \\ \partial\lambda/\partial s \end{bmatrix} = \begin{bmatrix} \partial\vec{R}/\partial s \\ \partial N/\partial s \end{bmatrix}$$

$$N = (c_i^{N+1} - c_i^N)^2 + (\lambda^{N+1} - \lambda^N)^2 - \Delta s^2$$

Initial guess for the new solution (N+1):

$$\vec{c}^{N+1} = \vec{c}^N - \frac{\partial\vec{c}^N}{\partial s} \Delta s$$

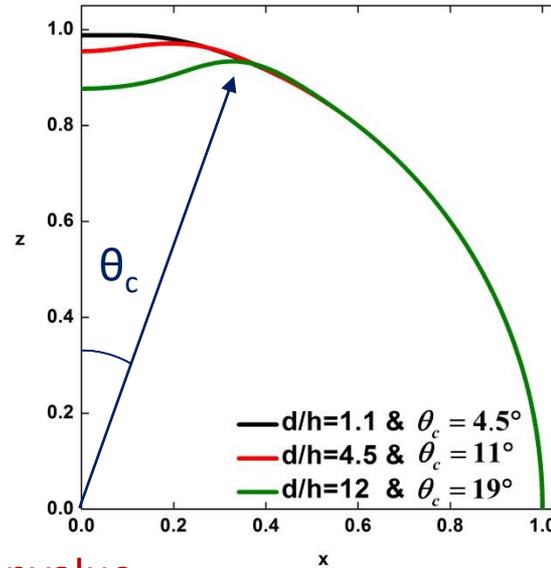
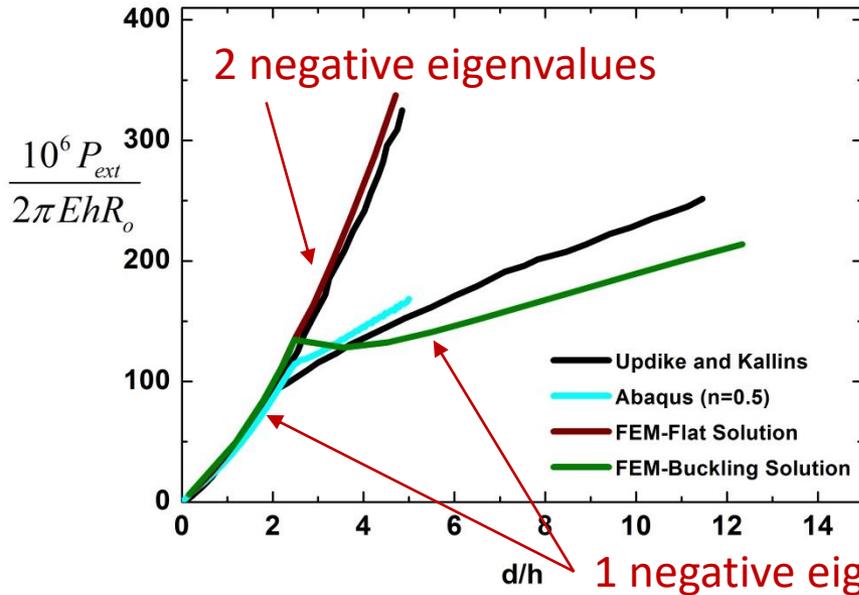
$$\lambda^{N+1} = \lambda^N - \frac{\partial\lambda^N}{\partial s} \Delta s$$



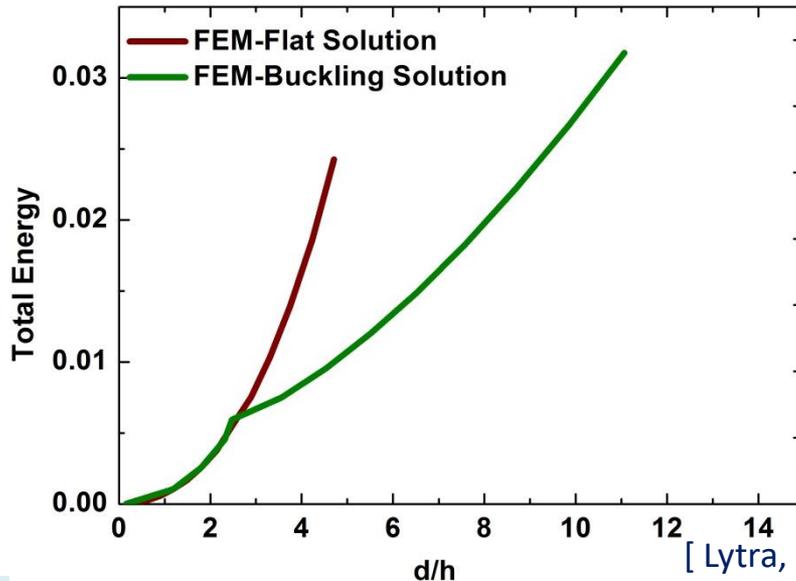
Initial Guess \longrightarrow Newton-Raphson Iterations \longrightarrow Convergence



Validation of Classic Contact Modeling



$E = 10^9 Pa$
 $R/h = 100$



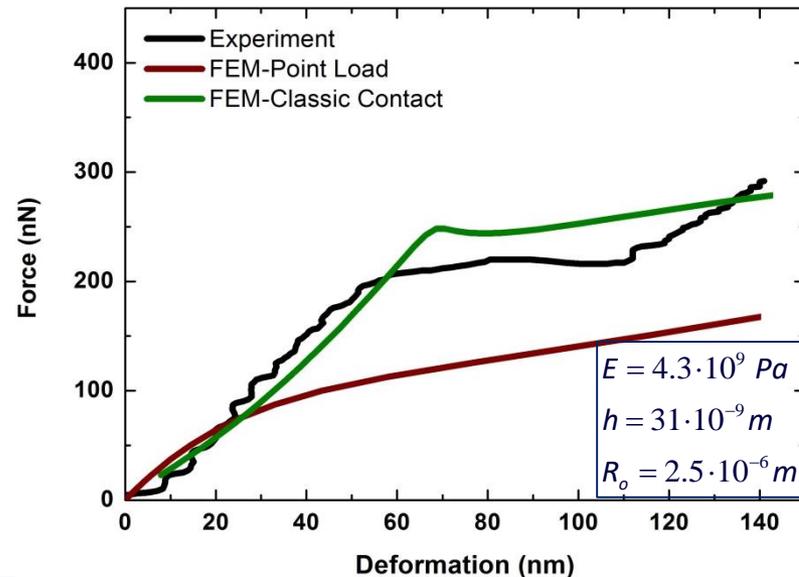
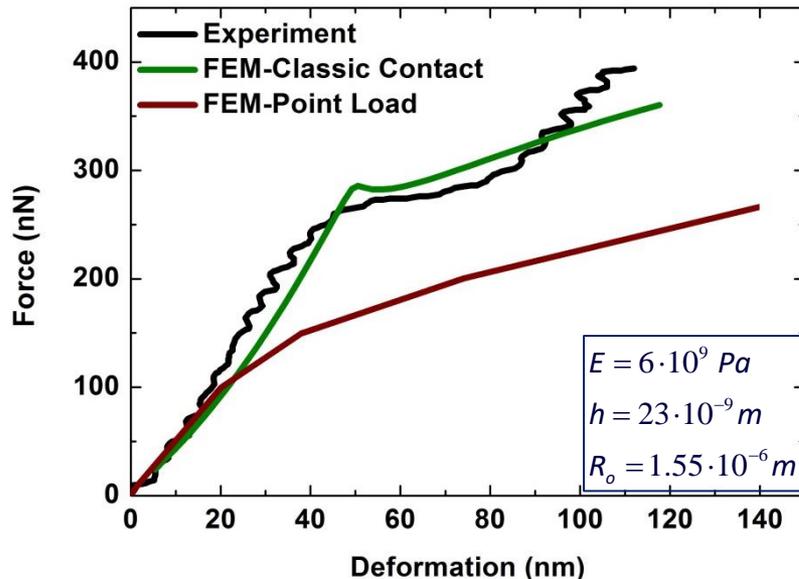
- Buckling point by FEM: (2.5, 130) & $\theta_c = 9^\circ$
- Buckling point by U&K: (2.1, 100) & $\theta_c = 8^\circ$
- The buckling FEM curve corresponds to a solution with the same number of negative eigenvalues with the flat curve before buckling.

[Updike & Kallins, *Trans. ASME*, 1970, 1972, 1972]

[Lytra, Pelekasis, Zafiropoulou, Zisis, Giannakopoulos, 8th GRACM Full Paper]



Classic Contact Modeling-AFM Data of Polymeric Bubbles



- Linear region - flat shape.
- Non-linear region - buckled-shell.
- The simulation can capture the experimental data.
- The amount of pre-stress explains the different response of experimental curves.
- The discrepancy in higher values of deformation is probably due to 3D-effects.

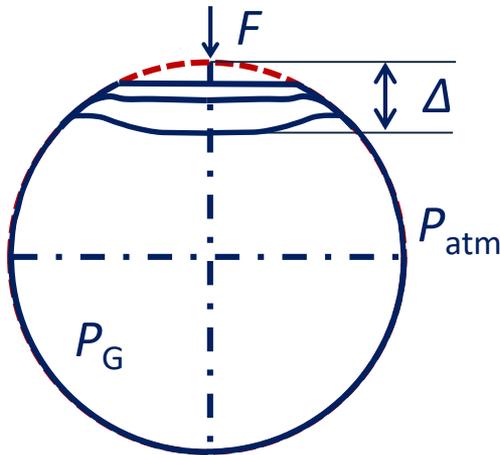
[Glynos et al., *Langmuir*, 2009]

[Lytra, Pelekasis, Sboros, Glynos, Koutsos, *Procedia IUTAM*, 2015]



Asymptotic Analysis for Polymeric Bubbles

Point Load



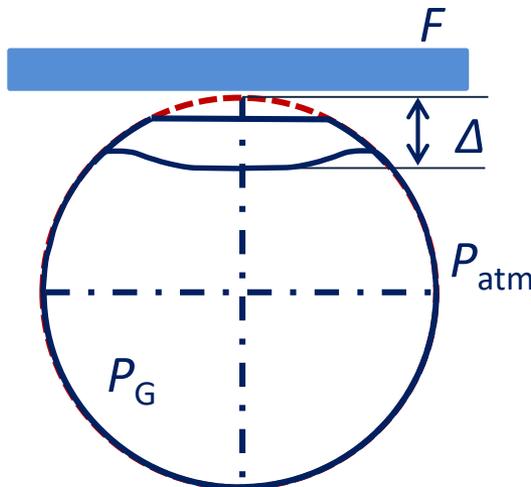
Reissner's eq. $F = \frac{4}{[3(1-\nu^2)]^{0.5}} \frac{Eh^2}{R_o} \Delta$

[Reissner, *J. Math. Phys.*, 1946]

Pogorelov's eq. $F = \left[\frac{3.56E^2h^5}{(1-\nu^2)R_o^2} \Delta \right]^{0.5}$

[Pogorelov, *Am. Math. Soc.*, 1988]

Plane Contact



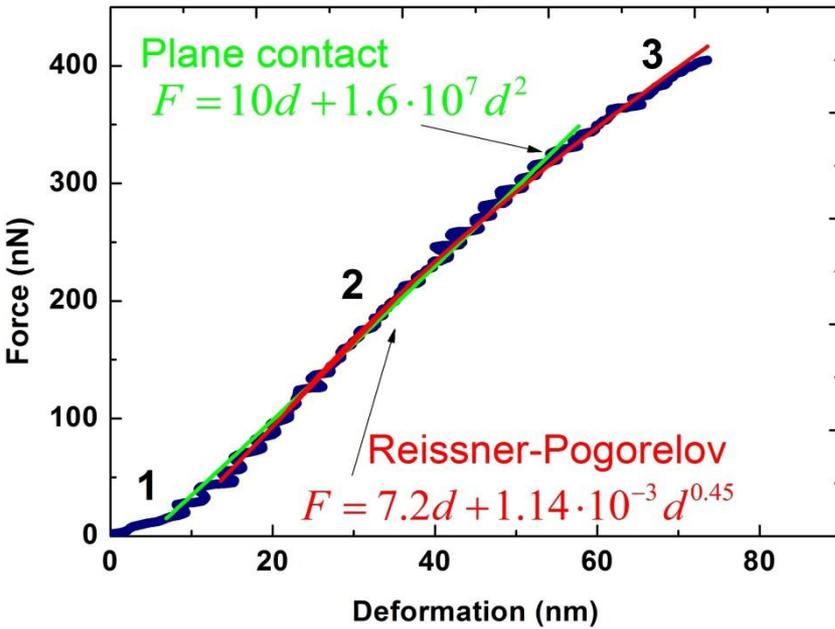
Stage I: $F = \frac{4}{[3(1-\nu^2)]^{0.5}} \frac{Eh^2}{R_o} \Delta + 0.06543 \frac{Eh}{R_o} \Delta^2$

Stage II: $F = 3.807 \frac{Eh^3}{(1-\nu^2)R_o} \left(\frac{\Delta\sqrt{1-\nu^2}}{h} \right)^{0.5}$

[Lytra, Pelekasis, Zafiropoulou, Zisis, Giannakopoulos, 8th GRACM Full Paper]



Elastic Properties Estimation - Polymeric Bubbles



Reissner's eq.

$$F = \frac{4}{[3(1-\nu^2)]^{0.5}} \frac{Eh^2}{R_o} \Delta$$

Pogorelov's eq.

$$F = \left[\frac{3.56E^2h^5}{(1-\nu^2)R_o^2} \Delta \right]^{0.5}$$

Plane Contact:

$$F = \frac{4}{[3(1-\nu^2)]^{0.5}} \frac{Eh^2}{R_o} \Delta + 0.06543 \frac{Eh}{R_o} \Delta^2$$

$R_o = 1.3 \mu\text{m} ; \nu = 0.5$	AFM Experiment	Reissner-Pogorelov	Plane Contact
Young's Modulus [GPa]	10-16	8.5	20
Shell Thickness [nm]	20	25	16

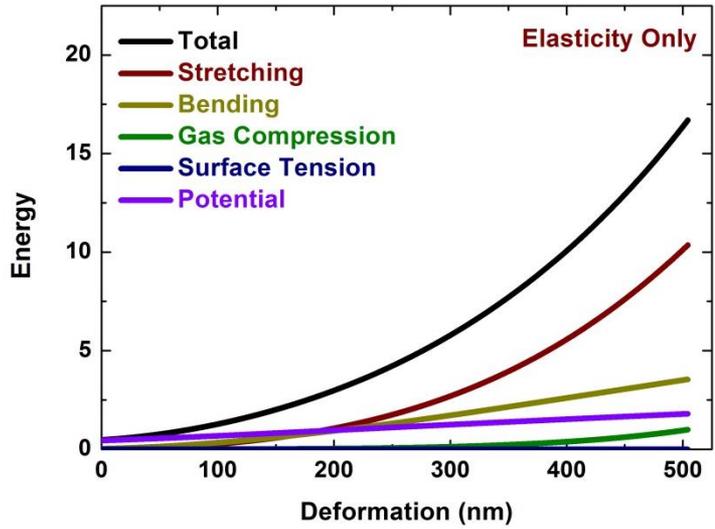
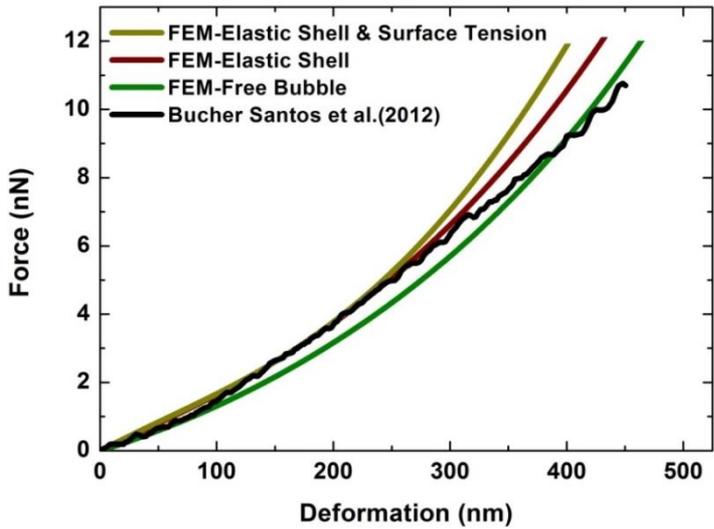
The elastic properties can be estimated by a single measurement, the f-d curve.

[Glynos et al., *Langmuir*, 2009]

[Lytra, Pelekasis, Sboros, Glynos, Koutsos, *Procedia IUTAM*, 2015]



Adhesive Potential-Bubble Covered With Lipid

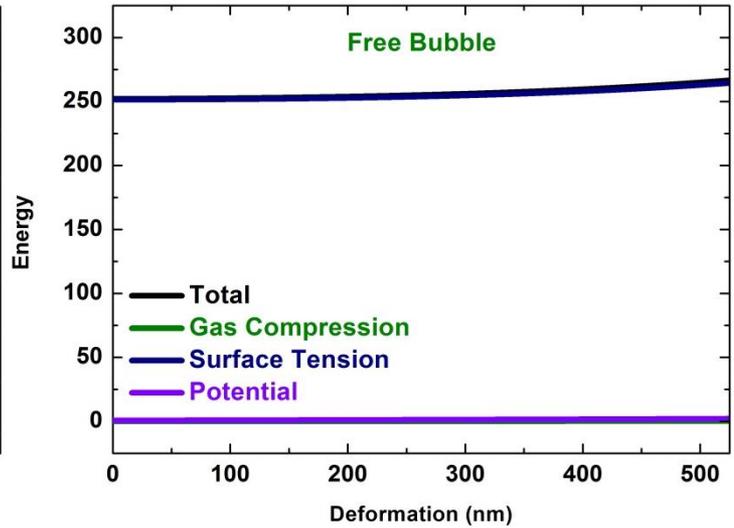
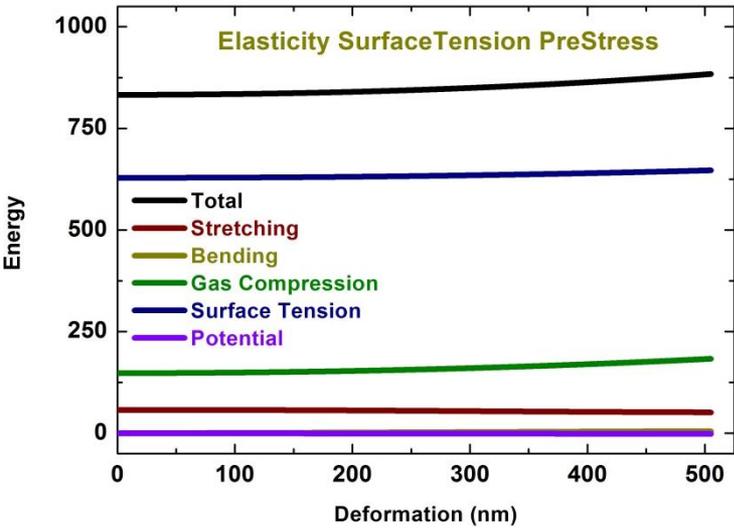


Elasticity Only

$\chi=0.05$ N/m
 $k_b=3 \cdot 10^{-16}$ Nm
 $W_o=10^{-4}$ N/m
 $\delta_A=50 \cdot 10^{-9}$ m
 $\gamma=1.07$
 Mooney-Rivlin (b=1)

Elast. Surf. Tension Pre Stress

$\chi=0.05$ N/m
 $k_b=3 \cdot 10^{-16}$ Nm
 $W_o=10^{-4}$ N/m
 $\delta_A=50 \cdot 10^{-9}$ m
 $\sigma=10^{-2}$ N/m
 $V=-0.14 \cdot 10^{-6}$ m
 $\gamma=1.07$
 Mooney-Rivlin (b=1)



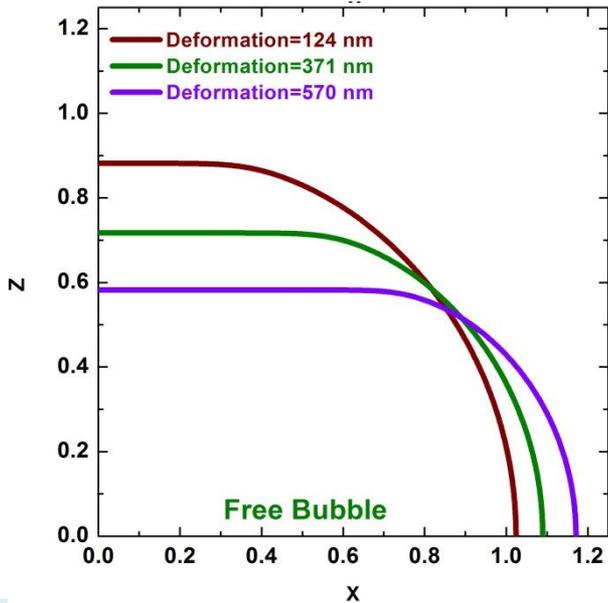
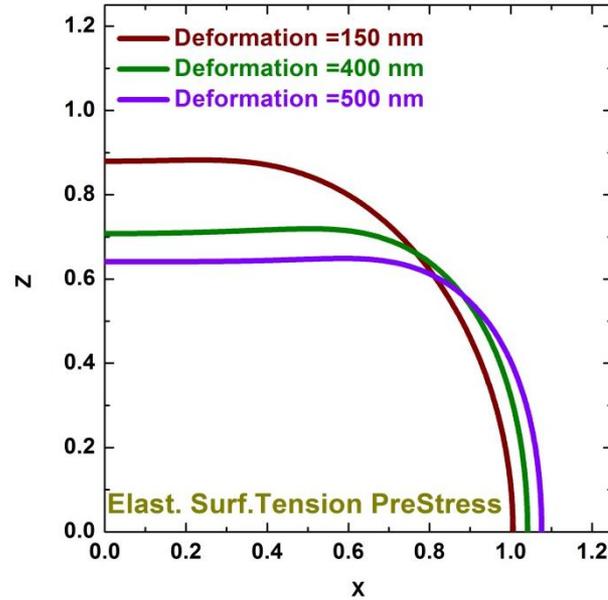
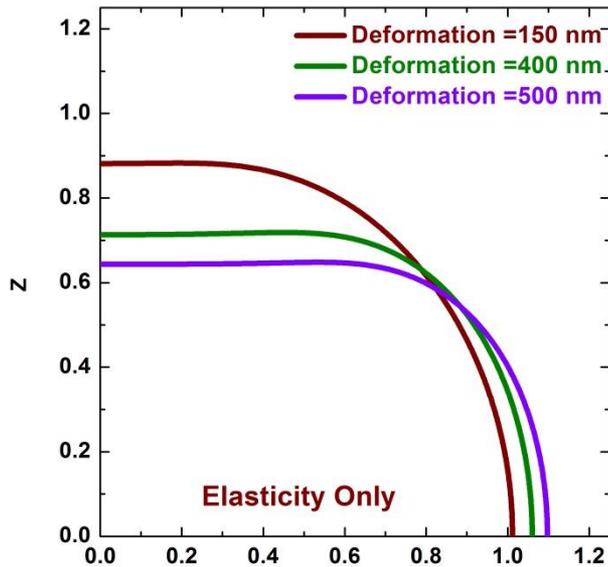
Free Bubble

$\sigma=4 \cdot 10^{-2}$ N/m
 $W_o=10^{-4}$ N/m
 $\delta_A=50 \cdot 10^{-9}$ m
 $\gamma=1.07$

[Bucher Santos et al., *Langmuir*, 2012]



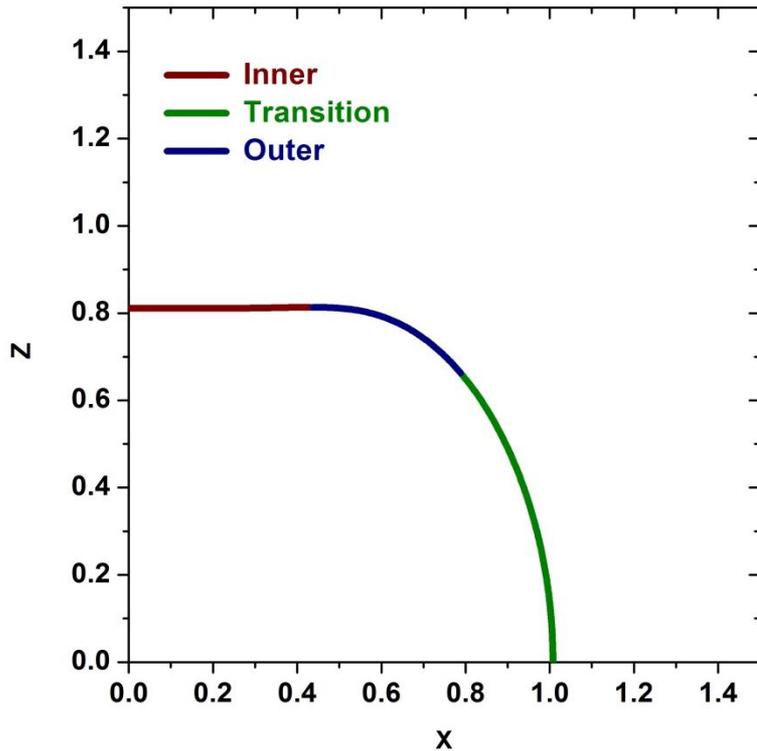
Adhesive Potential-Bubble Covered With Lipid



- The shell remains flattened.
- The contact angle is not constant.



Asymptotic Analysis for Lipids



Inner Region:

$$P_{in} - P_{out} \sim \frac{\partial W}{\partial n}$$

[Pressure difference \sim Disjoining Pressure]

Transition Region:

$$\frac{\partial W}{\partial n} \sim k_b$$

[Bending \sim Disjoining Pressure]

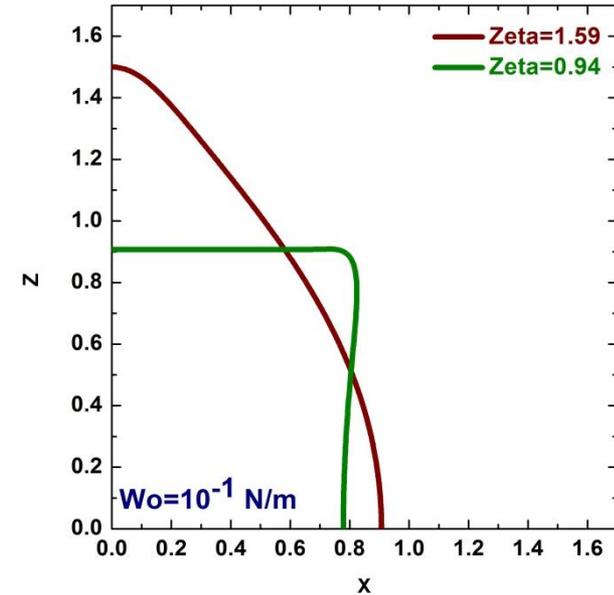
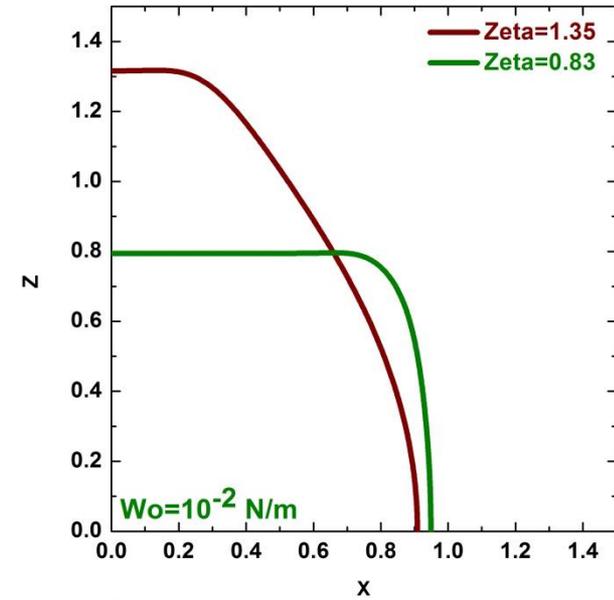
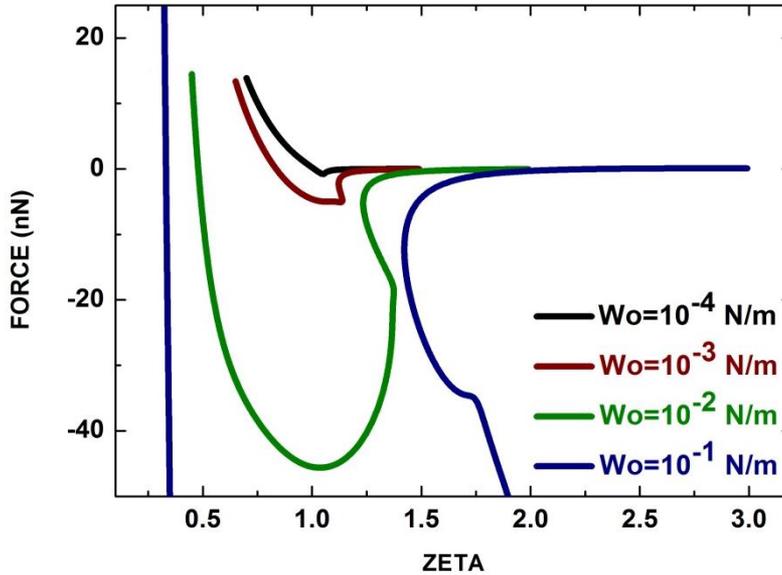
Outer Region

$$P_{in} - P_{out} \sim k_b + \chi$$

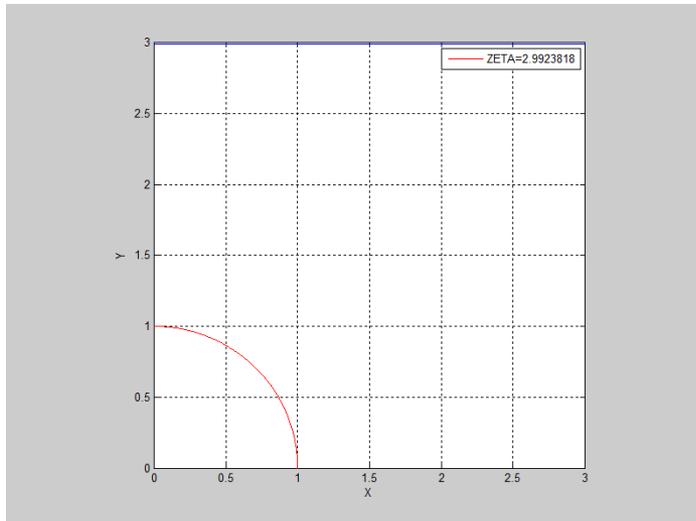
[Pressure difference \sim Bending and stretching]



Adhesive Potential-Parametric Study (Elasticity Model)



$W_o=10^{-1}$ N/m



- The classic contact model can easily simulate the force-deformation curve of a microbubble covered with polymeric shell.
 - The linear regime followed by a non-linear is associated with a sub-critical bifurcation.
 - In the linear regime, the shell remains flattened, but in the non-linear buckling takes place.

- In cases of lipids, which are softer, the intermolecular forces must be accounted for.
 - The experimental curve is almost linear, which indicate that buckling is not taking place.
 - Three models are considered in the present work, where no buckling was observed. The change of length is preferable, than bending, due to adhesion, which attaches the shell in cantilever.
 - The elasticity only model seems to be the more suitable for the simulations of the static response of CA microbubbles, covered with lipid.
 - The estimation of the elastic properties, requires further asymptotic analysis and perhaps different experimental set-up with side view photos.



Current & Future Work

- Simulation of a drop and a microbubble resting on a flat surface (Trapped microbubbles).
- Further asymptotic analysis and estimation of the elastic properties of bubbles covered with lipids.



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European Union
European Social Fund



MINISTRY OF EDUCATION & RELIGIOUS AFFAIRS
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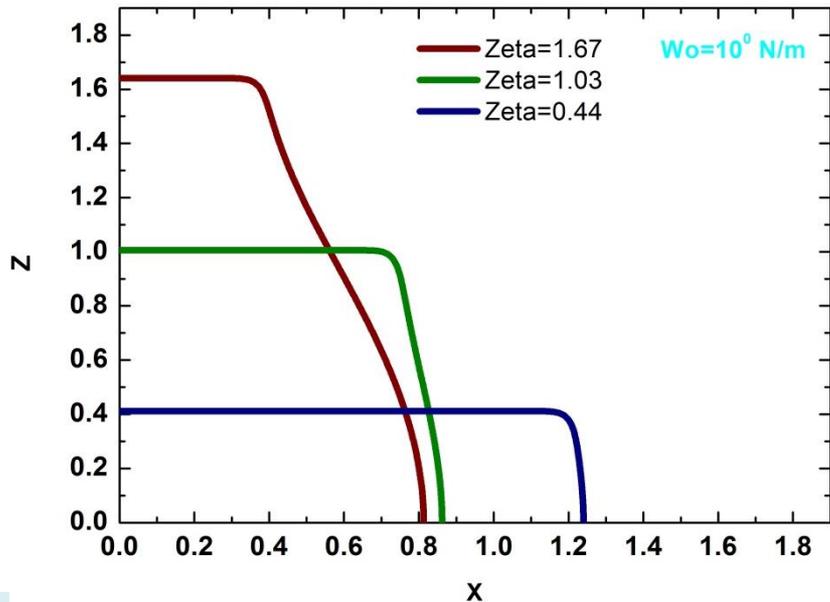
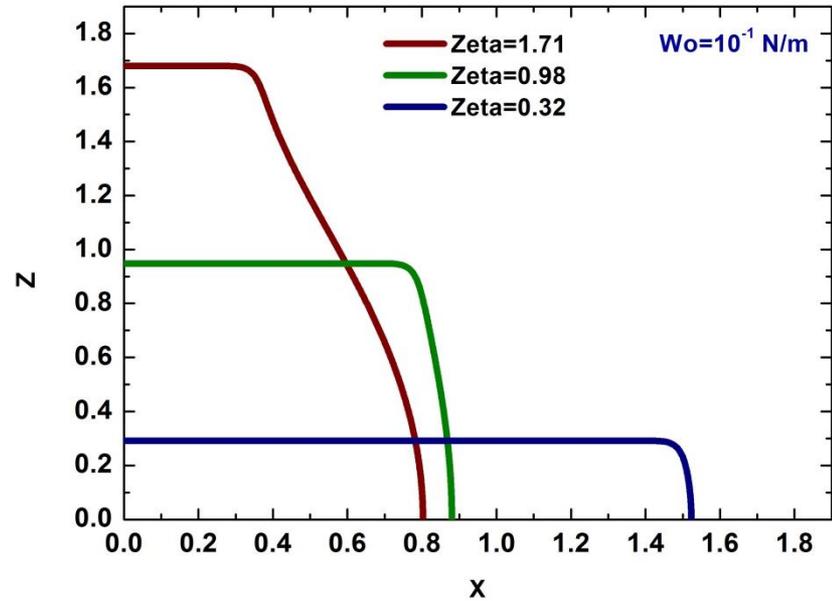
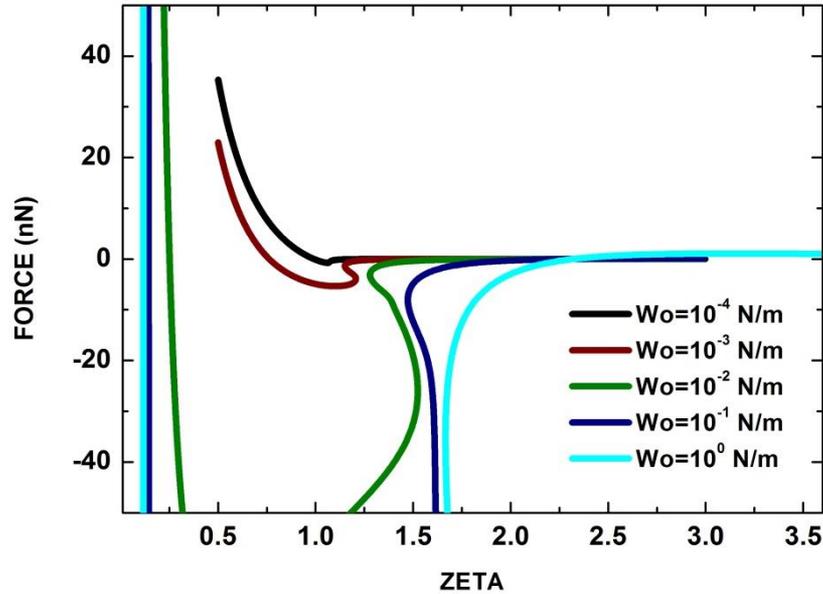


pel@uth.gr ; <http://contrast-aristeia.mie.uth.gr/>
<http://www.mie.uth.gr/>

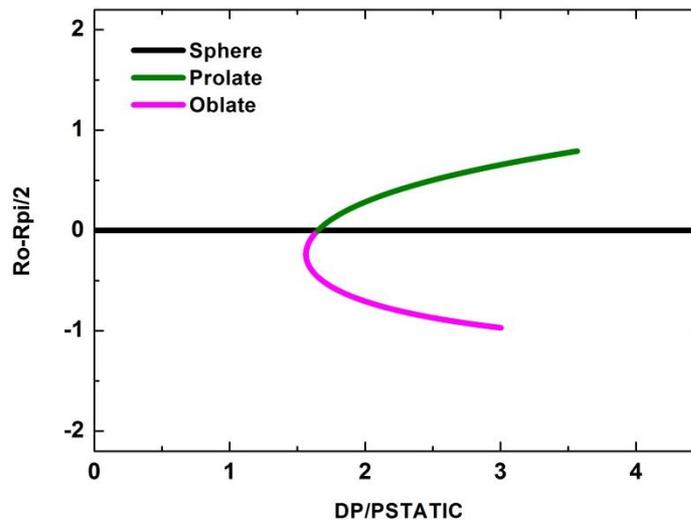
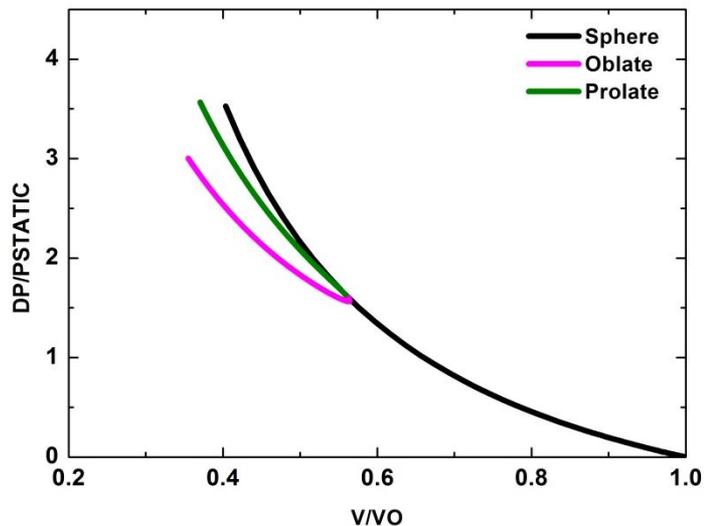
Thank you for your attention!
Ευχαριστώ πολύ!

Questions?

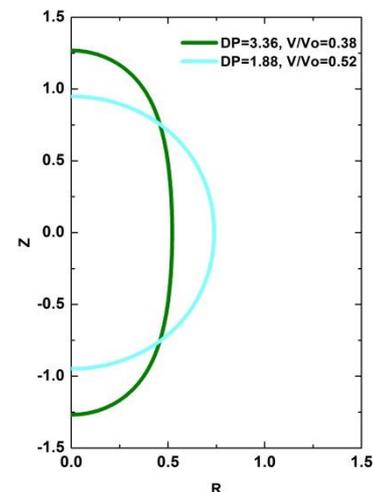
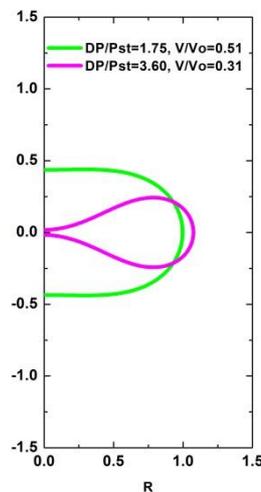
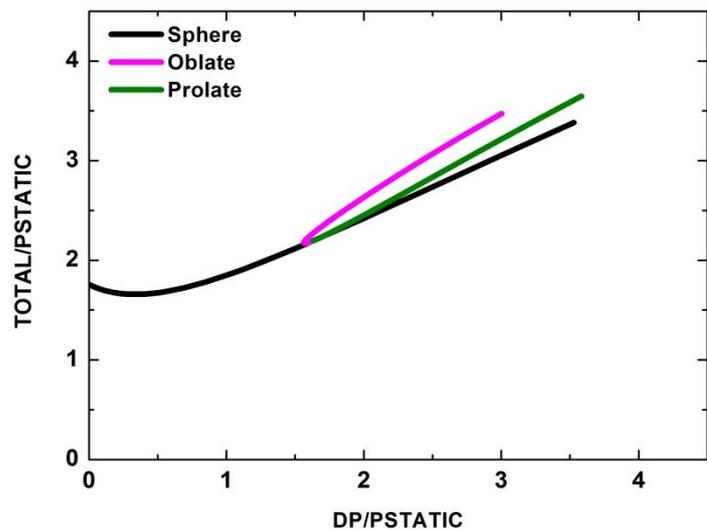
Adhesive Potential-Parametric Study (Free Bubble)



Bifurcation Diagrams



$G_s = 40$ MPa
 $k_b = 3 \cdot 10^{-14}$ Nm
 $\sigma = 0.051$ N/m
 Mooney-Rivlin

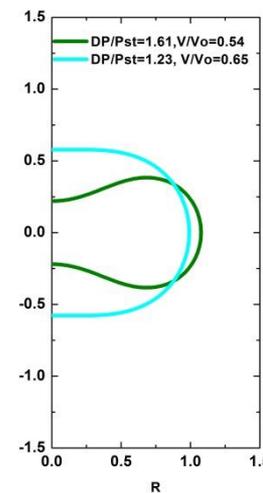
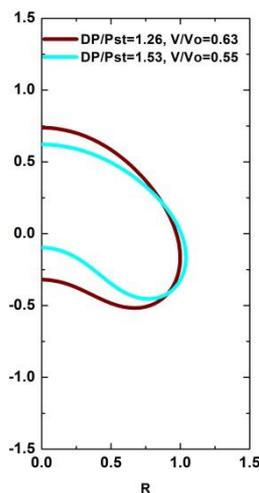
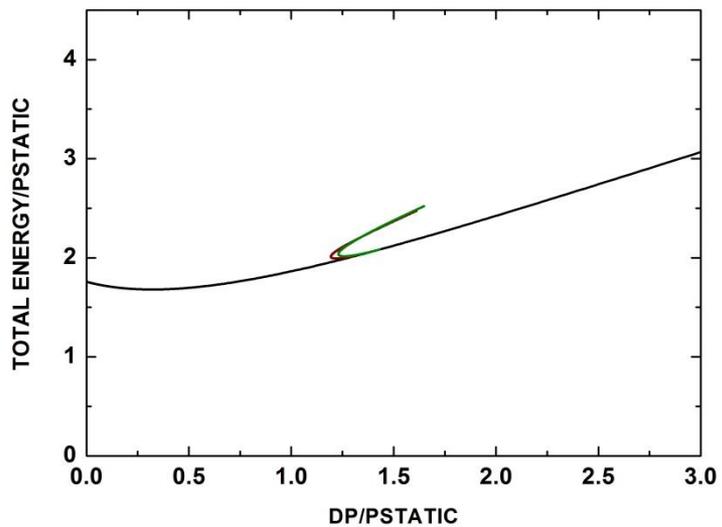
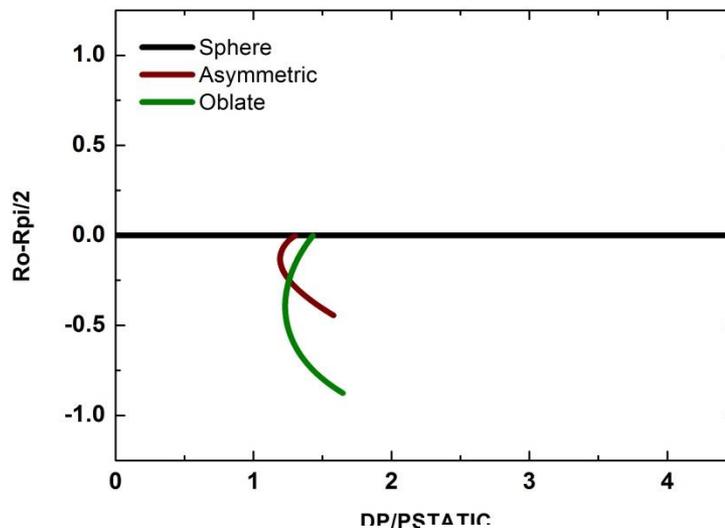
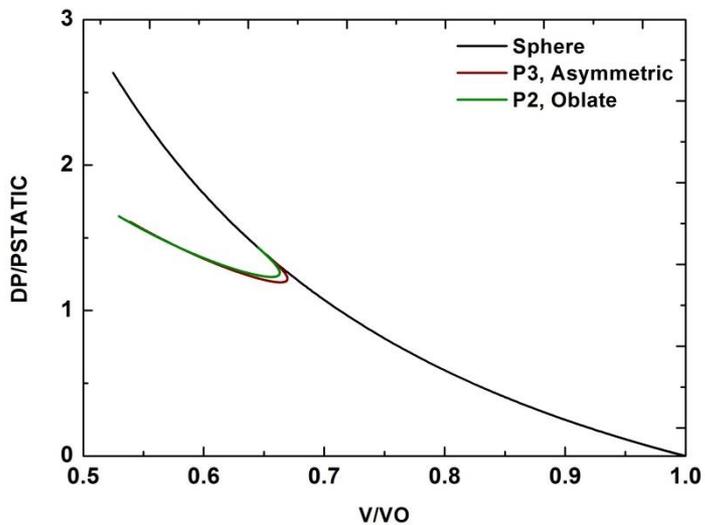


[Lytra & Pelekasis, *Fluid Dyn. Res.*, 2014]



Bifurcation Diagrams

$G_s = 80 \text{ MPa}$
 $k_b = 3 \cdot 10^{-14} \text{ Nm}$
 $\sigma = 0.051 \text{ N/m}$
 Mooney-Rivlin



[Lytra & Pelekasis, *Fluid Dyn. Res.*, 2014]



- The static response of microbubbles, covered with an elastic shell, subject to a uniform pressure was investigated.
 - In this problem, the shell remains spherical for increasing the external overpressure, but its volume is decreasing.
 - When the overpressure reaches the critical load, the shell buckles into a symmetric or a non-symmetric shape. The buckling is indicated by one more negative eigenvalue. Following simple continuation, the post-buckling f-d curve is generated.
 - In every post-buckling curve a limit point was accounted and arc-length continuation was performed.
 - In the case of $G_s=40$ MPa, the first bifurcation point is dominated by the a *symmetric* eigenmode (P2), which leads to a oblate or prolate shapes.
 - In the case of $G_s=80$ MPa, the first bifurcation point is dominated by a *non-symmetric* eigenmode (P3) and non-symmetric shapes.

